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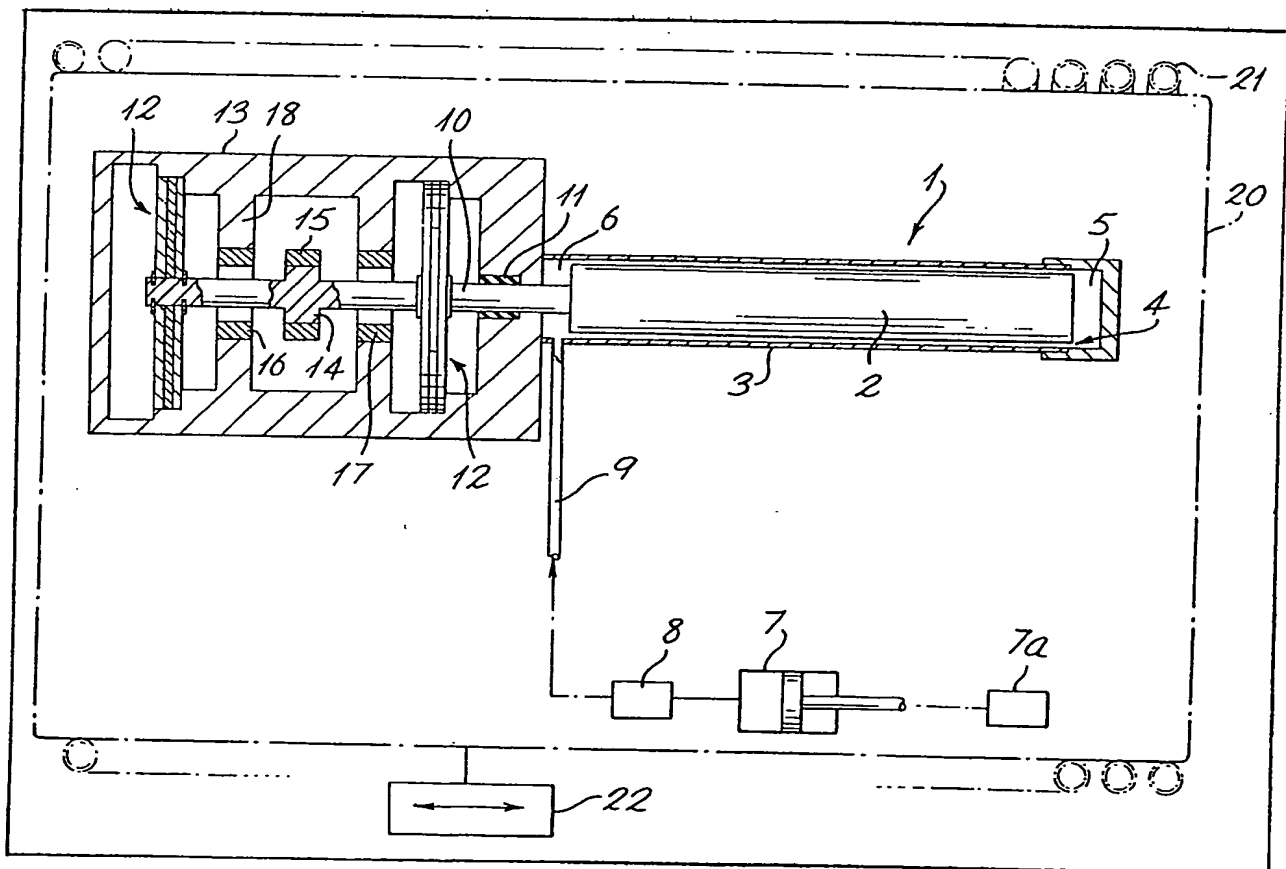
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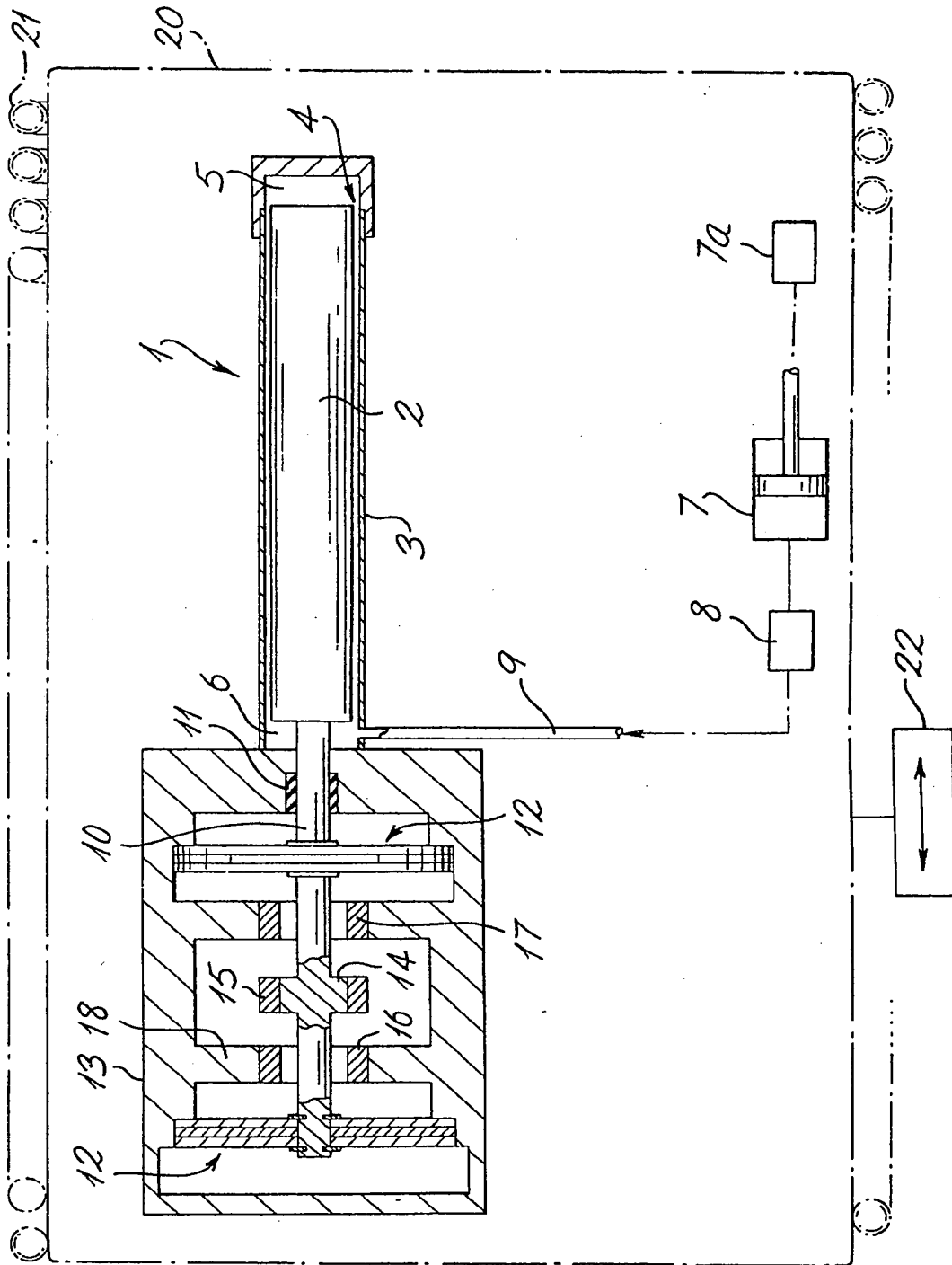
(54) Improvements in or relating to
free piston heat engines

(57) A Stirling cycle heat engine,
specially suitable for use in
environments subject to acceleration
or large temperature variations,
including a free and reciprocating
displacer piston (2) fitted with a non-
contact device which acts with
increasing force to oppose further
movement whenever the piston
overshoots desired limits of its
movement. The device may include a
magnet (15) which is carried by the
piston shaft (10) and which
experiences repulsion from like poles
of magnets (16, 17) carried by the
shaft housing (13).



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SPECIFICATION

Improvements in or relating to free piston heat engines

This invention relates to heat engines,

5 especially those employing the Stirling thermodynamic cycle. Stirling engines, as is well known, contain at least one of each of two essential moving parts, the movements of which are similar but must be out-of-phase with each other within certain limits. One of these parts is
10 usually known as the displacer, and often comprises a plunger or piston movable with clearance within a cylinder whereby to transfer a mass of gas in alternate directions between the
15 two ends of the cylinder. It is a characteristic of the cycle that one end of the displacer becomes or is maintained cold relative to the other, hence the use of Stirling engines (working as heat pumps) in refrigerators. The relatively hot end of the
20 displacer is connected by way of a heat exchanger to the other essential moving part of the engine, which typically comprises a piston movable within a cylinder and will be referred to as the compressor. This moving part constitutes the
25 interface between the machine and mechanical work: when the engine is acting as a heat pump the piston of this part is externally driven. If however the engine is to work in the reverse sense, then external power is used to maintain the
30 appropriate temperature difference between the two ends of the displacer. The resulting pulsations of pressure within the engine drive the piston of the compressor so that it can perform external mechanical work.

35 It is known for the displacer and compressor pistons of Stirling engines, and indeed for comparable moving parts of other heat engines, to be connected to rigid mechanical linkages that positively determine their exact positions at all
40 times. However it is also known for such pistons to be "free", that is to say to be suspended by fluid or mechanical springs so that their exact positions are not so determined. The present invention applies to heat engines having at least one free
45 piston as so defined, and especially to Stirling engines in which not only the displacer piston but also the compressor piston may be free. For instance the compressor piston may be connected to an electromagnetic device that acts as a motor
50 to drive the compressor when the engine is acting as a heat pump, and that acts as a generator driven by the compressor when it is acting in the reverse sense.

The need for the present invention is
55 demonstrated particularly by the type of Stirling cycle heat pump in which the piston of the displacer is specially "free", being neither positively driven nor linked to the movements of the compressor in any way other than through the
60 medium of the working fluid of the machine. In such a Stirling engine the displacer is simply so designed that its free response to the movements of the compressor, as reflected by changes in the velocity and pressure of the working fluid, is such

65 that it oscillates at the same frequency as the compressor but at an appropriate phase shift. The "Beal"-type machine is one known Stirling engine that works in this way. Such a machine has evident potential advantages in simplicity and
70 therefore in cost over those in which the movements of the displacer are either positively driven or subject to external control. What is more, the design of such "free response" mechanisms has reached the point where the optimum phase-
75 relationship between the movements of the compressor and displacer is obtained within close limits, so that the efficiency attainable with such mechanisms compares favourably with those in which the displacer is not free. However, Stirling
80 cycle engines find frequent use in cryogenic refrigerators and other plant in which the engine will be subjected to large variations of temperature, and this creates a problem for "free response" machines as just described. Having
85 arranged for the compressor and displacer strokes to be of the correct amplitude at say room temperature, as the temperature falls the reciprocating parts will tend to overshoot. This reduces performance, causes often unacceptable
90 noise and may lead to mechanical failure. The tendency is especially great if the equipment containing the engine is not stationary but is subject to acceleration.

The present invention aims to provide a simple,
95 maintenance-free way of countering any such tendencies for the amplitude of the stroke of "free" pistons to change in such conditions. According to the invention a free piston of a heat engine includes a non-contact device which acts with
100 increasing force to oppose further movement whenever the piston overshoots predetermined limits of its reciprocatory movement. The device may be of magnetic type, giving rise to forces of repulsion between like magnetic poles. For
105 instance the piston may carry a magnet, mounted for example on the piston shaft, and two magnets may be fixed to the shaft housing so that a first pole of the moving magnet approaches a like pole of one of the fixed magnets when the piston tends
110 to overshoot in one direction, and the second pole of the moving magnet approaches a like pole of the second fixed magnet when there is overshoot in the opposite direction. The magnets should be of Samarium Cobalt or other type that will
115 withstand strong demagnetising fields. The piston may be the compressor piston, or more especially the displacer piston, of a Stirling cycle engine, and the heat engine may be located in an environment where it may be subject to large variations of
120 temperature and/or to acceleration.

The invention will now be described, by way of example, with reference to the accompanying drawing which is a diagrammatic axial section through a Stirling engine displacer.

125 The displacer 1 comprises a free piston 2 mounted to reciprocate within a cylinder 3 from which it is separated by a small annular clearance 4. The walls of this clearance act as a regenerative heat exchanger, and movement of the piston to

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and from within the cylinder causes gas to be displaced through clearance 4 in alternate directions between the blind or distal end 5 of the cylinder and the opposite end 6. Such movement
 5 results from the free response of piston 2 to the movements of the free piston (connected to an electromagnetic device 7a) of the compressor 7, those movements being reflected in movement of the working gas of the machine which reaches
 10 end 6 of the displacer by way of a heat exchanger 8 and a conduit 9. Oscillation of piston 2 in response to the movements of the piston of the compressor 7 causes end 5 to become relatively cold and end 6 relatively warm,
 15 so that the machine acts as a heat pump and end 5 may be used as the power source of a refrigeration unit.

Piston 2 carries a shaft 10 which passes through a gas-tight seal 11 and carries two flat spiral springs 12 by which it is mounted within a fixed housing 13. The springs 12 flex readily in the axial direction but are very stiff radially and so hold rod 10 and piston 2 accurately to axial reciprocation.

25 Shaft 10 carries a boss 14 around which a circular magnet 15 is mounted, and two similar magnets 16, 17 are mounted on flanges 18 projecting inwardly from the wall of housing 13, so that magnet 16 lies axially to one side of magnet 15 and magnet 17 lies axially to the other side. The polarity of magnets 15, 16 and 17 is arranged with like poles adjacent, so that as magnet 15 approaches either of the other two it is opposed by an increasing repulsive force, so
 35 opposing any tendency of piston 1 to overshoot its proper amplitude of movement as a result, for example, of a change in the temperature or of acceleration to which displacer 1 has been subjected. The means whereby the engine may be
 40 subjected to changes of temperature or to acceleration are illustrated diagrammatically: the engine is shown as being mounted within a container 20 wound with refrigerating coils 21, and connected to a prime mover 22.

45 As shown in the drawing, as magnet 15 approaches either of the other two then the force of repulsion that it experiences will vary in an

inverse manner relative to the distance between them. The force-distance curve depends on many
 50 factors including the shape of the magnets, their length-to-pole area, the ratio between the size of the pole faces and the distance between repelling magnets, etc. Hence it is possible to alter the damping characteristics of the system within wide
 55 limits by altering one or more of such geometrical features.

CLAIMS

1. A heat engine having a free reciprocating piston associated with a non-contact device which
 60 acts with increasing force to oppose further movement whenever the piston overshoots predetermined limits of its reciprocatory movement.

2. A heat engine according to Claim 1 in which the non-contact device is of magnetic type, giving rise to forces of repulsion between like magnetic poles.

3. A heat engine according to Claim 1 in which the piston carries a magnet mounted on the piston shaft, and in which two magnets are fixed to the shaft housing so that a first pole of the moving magnet approaches a like pole of one of the fixed magnets when the piston tends to overshoot in one direction, and the second pole of the moving
 70 magnet approaches a like pole of the second fixed magnet when there is overshoot in the opposite direction.

4. A heat engine according to Claim 1 adapted to work in accordance with the Stirling thermodynamic cycle, and in which the free piston is the displacer piston.

5. A heat engine according to Claim 1 adapted to work in accordance with the Stirling thermodynamic cycle, and in which the free piston is the compressor piston.

6. A heat engine according to Claim 1, mounted in an environment where it is subject to acceleration and/or to large variations of temperature.

7. A heat engine according to Claim 1, substantially as described with reference to the accompanying drawing.

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